



معهد الكويت للأبحاث العلمية
Kuwait Institute for Scientific Research

Modeling of Co-Current Crude Oil-Gas Flow in a Commercial Pipeline

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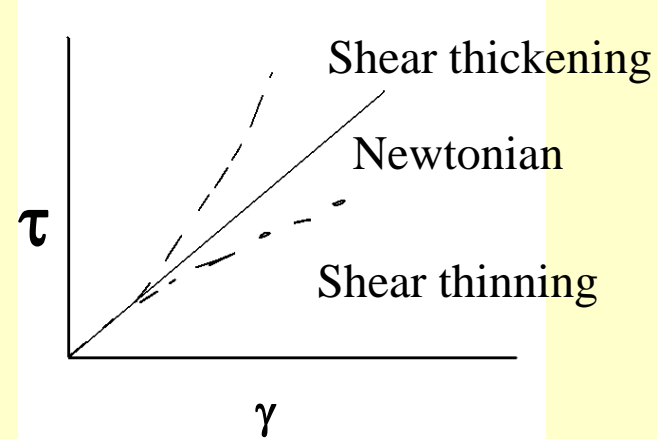
17th November, 2013

Outline

- **Examining the proposed model for two-phase flow**
- **Commercial Application**
- **Examine the adequacy of the model for real data of Crude Oil-Gas Pipeline**
- **Main conclusion and recommendations**

Homogeneous Flow Model

- **Newtonian Flow** ($\tau = -\mu\gamma$)
- **Non-Newtonian**
Viscoelastic
Pseudoplastic



Bingham plastic ($\tau = \mu_0 - \mu\gamma$)

Power law ($\tau = -k\gamma^n$)

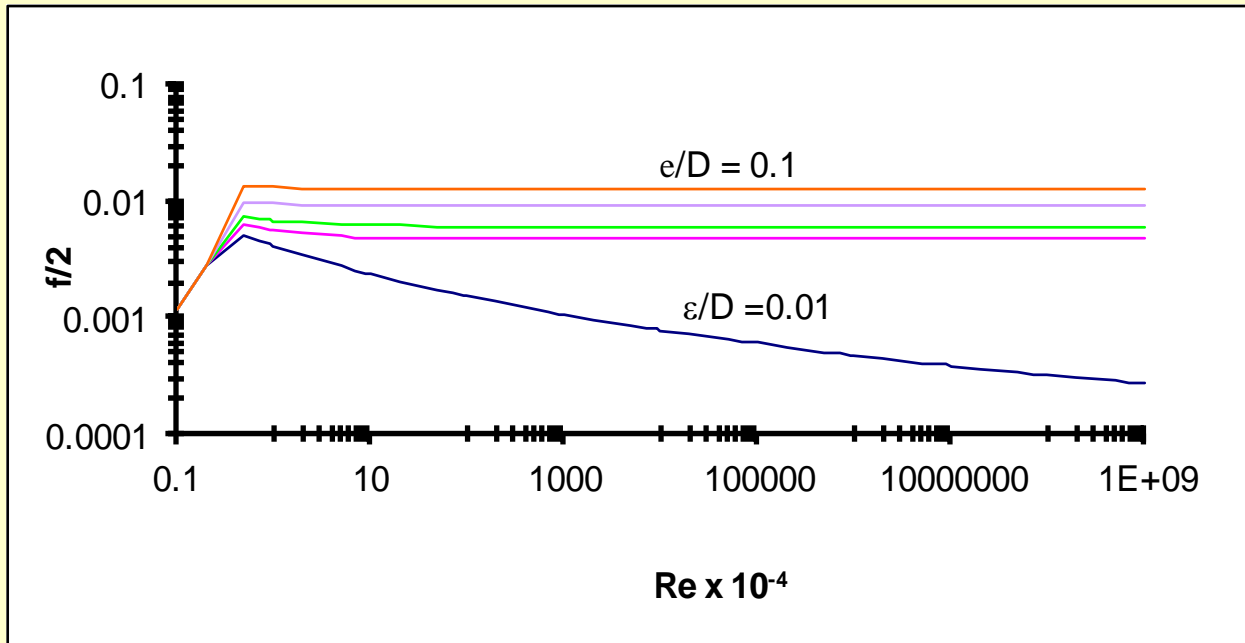
Herschel Bulkley Model ($\tau = \mu_0 - k\gamma^n$)

Pseudo-Homogeneous Model

Churchill

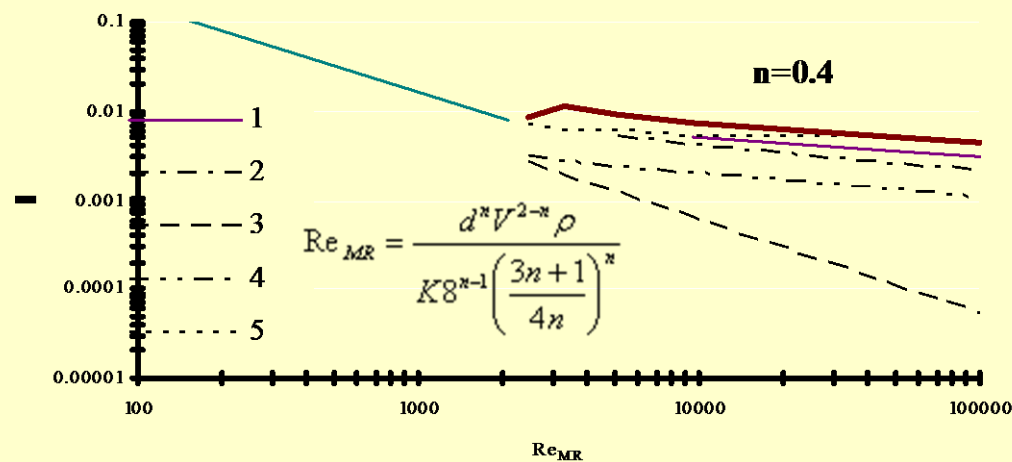
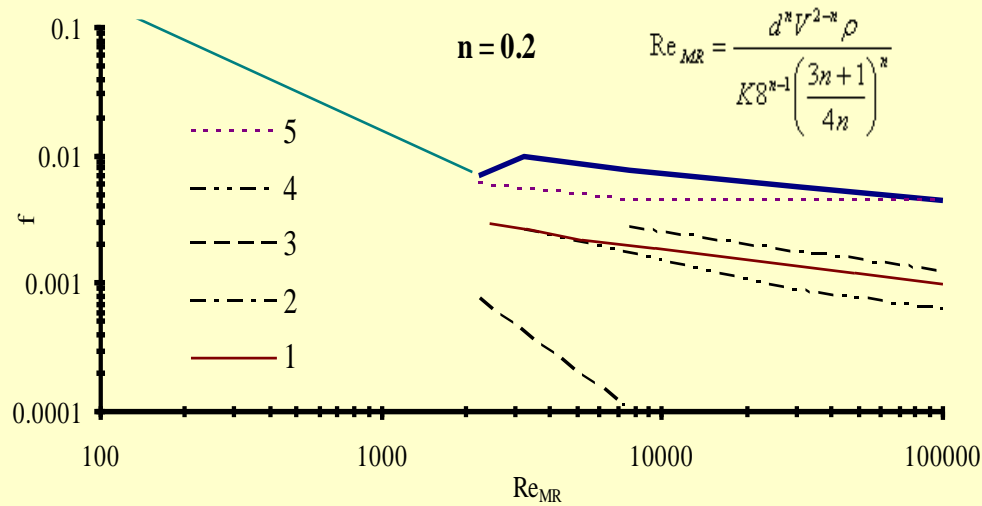
Eqn. $f = 12 \sqrt{\left(\frac{8}{\text{Re}^{12}} + \sqrt{(A + B)^3} \right)}$

$$A = \left[-2.457 \ln \left(\frac{7}{\text{Re}^{0.9}} + 0.27 \frac{\varepsilon}{D} \right) \right]^{16} \quad \text{and} \quad B = \left(\frac{37530}{\text{Re}} \right)^{16}$$



Non-Newtonian (Power Law)

5. Kembluski et al. (1962), 4. Thomas (1978), 3. Shaver & Merrill (1959), 2. Tomita (1959), 1. Metzner & Reed (1955)



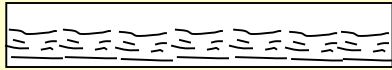
Study Objective

The aim of this study is to check the published models in the literature for two phase flow based on small laboratories experimental setups for air-water flow in 1” or 2” (25 mm, 50 mm) diameter pipes

- 1. To determine the characteristics of the specific Crude Oil**
- 2. Formulating the problem by dividing total submarine pipeline of 42 km length into n continuous segments**
- 3. Pressure drop calculation in each segment, knowing only the discharge pressure.**
- 4. Applying Flash calculation for each segment to know the gas generated that equilibrate with flowing crude at given Pressure**

Two-phase flow and Flow pattern

- **Stratified flow**

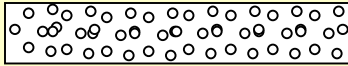


$$\chi = \sqrt{\frac{-\Delta P_G}{-\Delta P_L}}, \quad \phi_G = \sqrt{\frac{-\Delta P_{TP}}{-\Delta P_G}}, \quad \phi_L = \sqrt{\frac{-\Delta P_{TP}}{-\Delta P_L}}$$

- **Plug flow**



- **Bubbly flow**

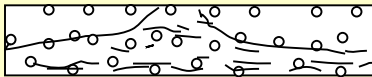


$$\phi_G^2 = \lambda^{0.205} \quad \text{for } 0.6 < \lambda < 1$$

$$\phi_G^2 = 1 - 0.031\lambda^{-2.25} \quad \text{for } 0.35 < \lambda < 0.6$$

$$\phi_G^2 = 1.95\lambda \quad \text{for } 0.05 < \lambda < 0.35$$

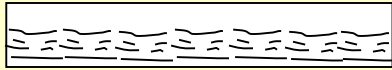
- **Segregated flow**



$$\text{where } \lambda \text{ is } \left(\frac{V_{SL}}{V_{SLC}} \right)^{1-n}$$

Two-phase flow and Flow pattern

- **Stratified flow**



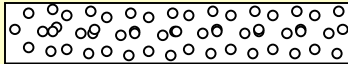
$$\chi = \sqrt{\frac{-\Delta P_G}{-\Delta P_L}}, \quad \phi_G = \sqrt{\frac{-\Delta P_{TP}}{-\Delta P_G}}, \quad \phi_L = \sqrt{\frac{-\Delta P_{TP}}{-\Delta P_L}}$$

- **Plug flow**



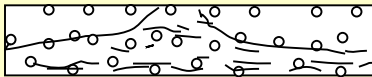
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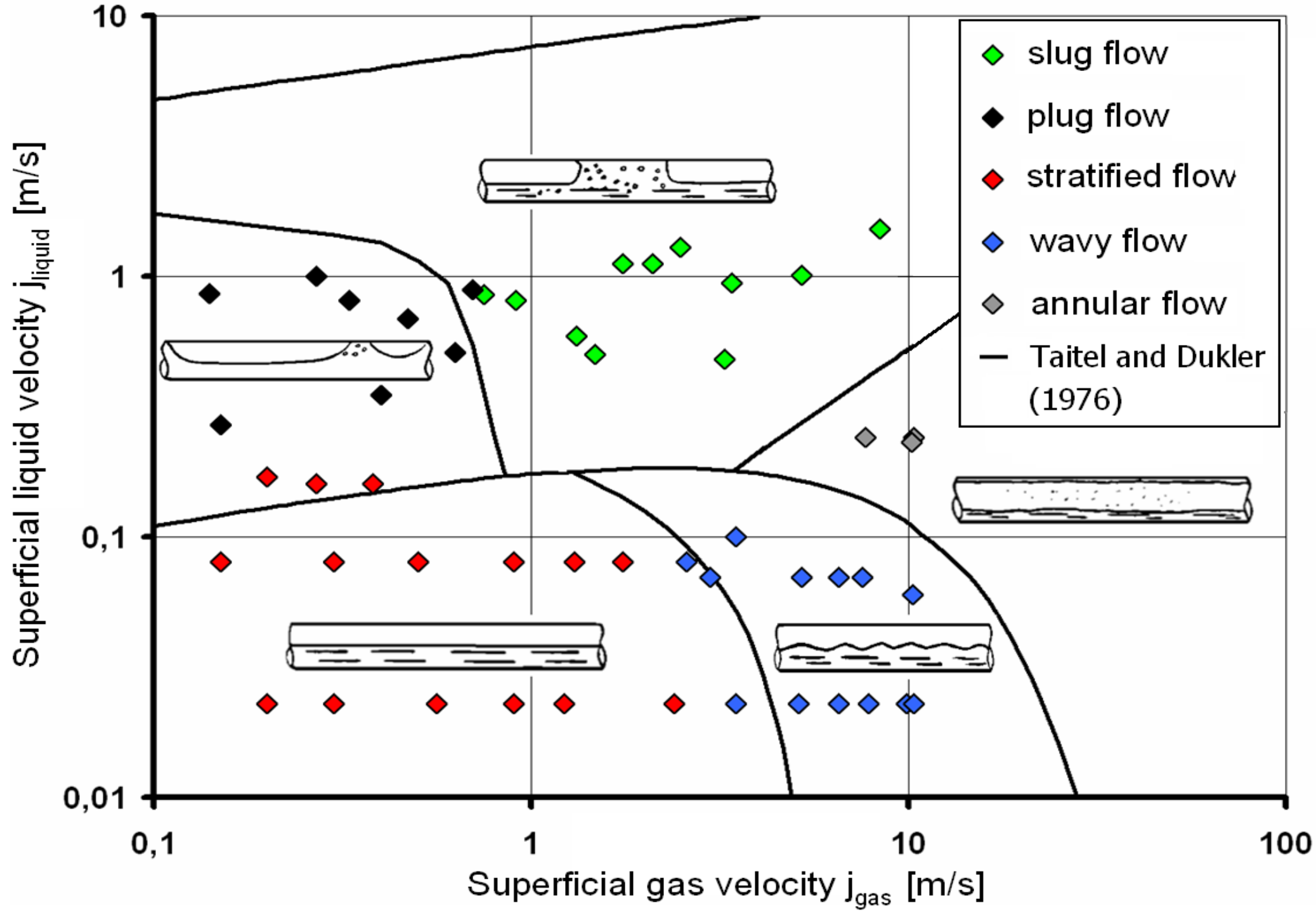
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SEGREGATED FLOW



Stratified



Wavy

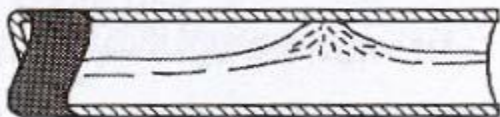


Annular

INTERMITTENT FLOW



Plug



Slug

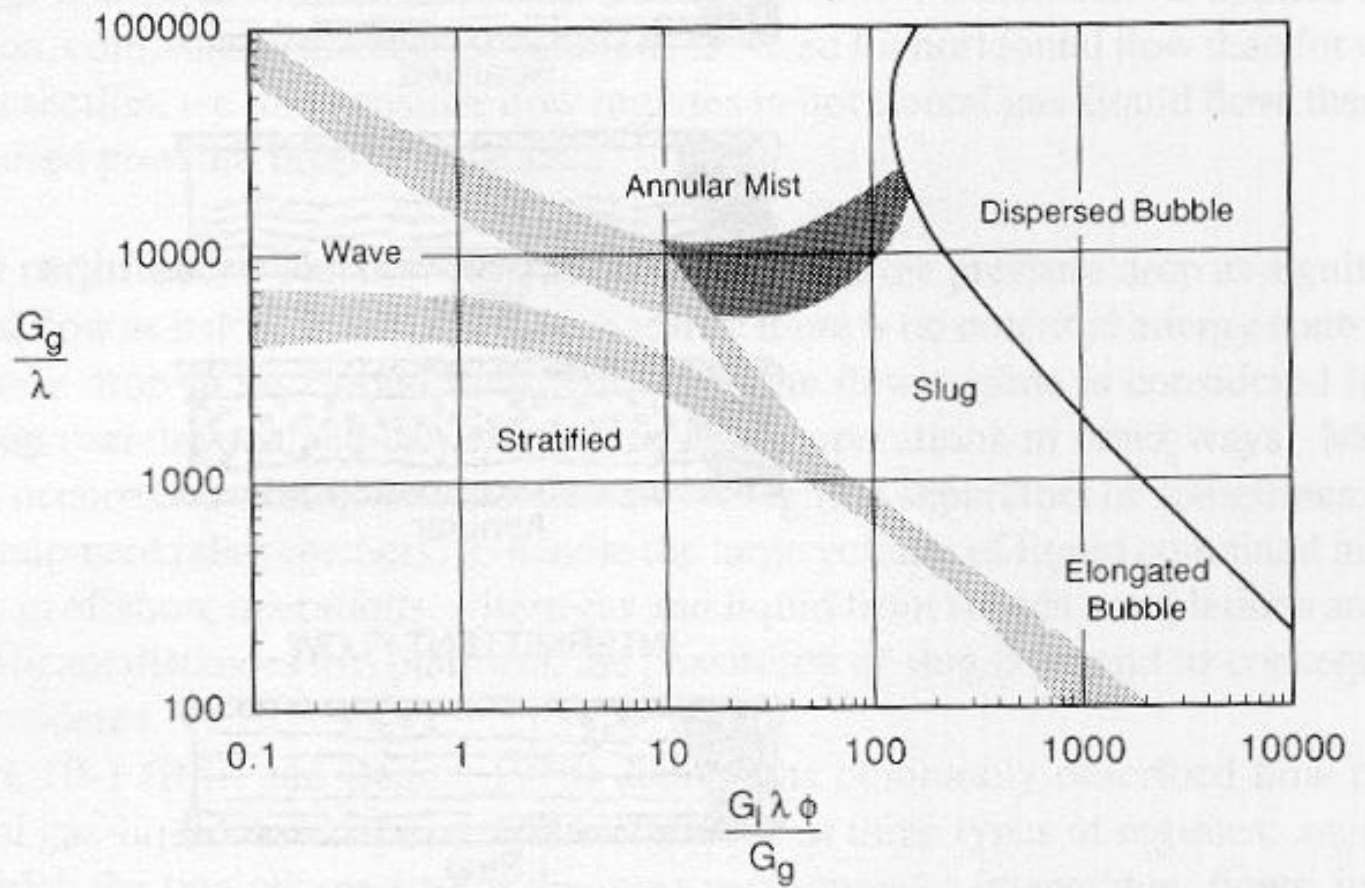
DISTRIBUTIVE FLOW



Bubble



Mist



Multistage separation Test

Stage No	First		Second	
Pressure (kPa)	2171	2171	517	517
Temp. (°C)	41	41	38	38
Method	measured	Flash Calc.	measured	Flash Calc.
Component	Mole Fraction	Mole Fraction	Mole Fraction	Mole Fraction
H ₂ S	0.00009	0.0013	0.0002	0.0029
N ₂	0.01	0.0099	0.0023	0.0025
CO ₂	0.0187	0.0168	0.0228	0.021
CH ₄	0.7296	0.7062	0.5116	0.4136
C ₂ H ₆	0.1527	0.163	0.2759	0.1913
C ₃ H ₈	0.0632	0.0725	0.1361	0.0138
i-C ₄ H ₁₀	0.0041	0.0046	0.0088	0.0484
n- C ₄ H ₁₀	0.0131	0.0157	0.027	0.0078
i-C ₅ H ₁₂	0.0022	0.0025	0.0041	0.0114
n- C ₅ H ₁₂	0.0032	0.0037	0.0059	0.0059
C ₆ -Group	0.0022	0.0024	0.0038	0.007
C ₇ Group	0.0006	0.0008	0.0009	0.0023
C ₈ -Group	0.0002	0.0004	0.0	0.001
C ₉ -Group	0.0001	0.0	0.0001	0.0002
C ₁₀ -Group	0.0001	0.0	0.0001	0.0001
C ₁₁ -Group	0.0	0.0	0.0	0.0
Total	1.00	1.00	1.0	1.0

Comparison

Pressure (kPa)	Temperature (°C)	Gas/Oil Ratio Measured	Gas/Oil Ratio calc.	Abs. % error
2171	41	312	324	3.84
517	38	78	95	21.7
103	32	93	70	24.73
Mean				16.8

Calculation of Thermo-physical properties

Thermo-physical properties were calculated for oil and gas over pressure range 1790-687 kPa using Hysys version 3.1 The UIQUAC equation of state was used.

$$\rho_L (kg m^{-3}) = -6.605 \times 10^{-3} P + 839.07$$

$$\rho_G (kg m^{-3}) = 7.7 \times 10^{-3} P + 1.82$$

$$\mu_L (Pa s) = -8.139 \times 10^{-7} P + 0.0059$$

$$\mu_G (Pa s) = 6.657 \times 10^{-6} P^{0.0714}$$

$$\sigma_L (N m^{-1}) = 2.999 \times 10^{-10} P^2 - 2.223 \times 10^{-6} P + 0.022$$

$$\frac{Q_G}{Q_{inlet}} = 11.091 \left(\frac{P_i}{P_{inlet}} \right)^4 - 37.605 \left(\frac{P_i}{P_{inlet}} \right)^3 + 49.352 \left(\frac{P_i}{P_{inlet}} \right)^2 - 30.91 \left(\frac{P_i}{P_{inlet}} \right) + 8.09$$

Pressure Drop Models

Total ΔP per unit length is sum of frictional, gravitational and accelerational

$$-\left(\frac{dP}{dz}\right)_T = -\left(\frac{dP}{dz}\right)_f - \left(\frac{dP}{dz}\right)_g - \left(\frac{dP}{dz}\right)_a$$

Beggs & Brill (1973)

$$-\left(\frac{dP}{dz}\right) = \frac{f_\phi G_m v_m}{2g_c d} \quad \text{and} \quad f_\phi = f_{ns} \cdot e^S$$

$$\lambda = \frac{Q_L}{Q_L + Q_G} \quad \text{and} \quad y = \frac{\lambda}{[H_L(\theta)]^2} \quad \text{For } 1 < y < 1.2, \quad S = \ln(2.2y - 1.2)$$

$$S = \left[\frac{\ln(y)}{-0.0523 + 3.182 \ln(y) - 0.8725 [\ln(y)]^2 + 0.01853 [\ln(y)]^4} \right]$$

$$f_{ns} = \left[2 \log \left(\frac{\text{Re}_{ns}}{4.5223 \log(\text{Re}_{ns}) - 3.8215} \right) \right]^{-2} \quad \text{and} \quad \text{Re}_{ns} = \frac{[\rho_L \lambda + \rho_G (1 - \lambda)] v_m d}{[\mu_L \lambda + \mu_G (1 - \lambda)]}$$

1. Continued

Mukherjee & Brill (1985)

$$-\left(\frac{dP}{dz}\right) = \frac{f_{ns} v_m^2 \gamma_m}{2gd} \quad \text{and } \gamma_m = H_L \gamma_L + H_g \gamma_g$$

$$f_{ns} = \left[2 \log \left(\frac{\text{Re}_{ns}}{4.5223 \log(\text{Re}_{ns}) - 3.8215} \right) \right]^{-2} \quad \text{and } \text{Re}_{ns} = \frac{[\rho_L \lambda + \rho_G (1 - \lambda)] v_m d}{[\mu_L \lambda + \mu_G (1 - \lambda)]}$$

$$H_L = \frac{A_L}{A} = \frac{1}{2\pi} (\delta - \sin(\delta)) \quad \text{where } \delta = 2 \cos^{-1} \left(1 - 2 \frac{h_L}{d} \right)$$

Oliemans¹ &²(1976)

$$-\left(\frac{dP}{dz}\right)_{f,tp} = \frac{f_\varphi G_\varphi^2}{2\rho_\varphi d_{\text{eff}}} \quad \text{where } G_\varphi = \frac{1}{[1 - (H_L - \lambda)] A_p} w_t, \quad D_{\text{eff}} = \sqrt{1 - (H_L - \lambda)} d$$

$$\rho_\varphi = \rho_L \frac{\lambda}{1 - (H_L - \lambda)} + \rho_g \frac{1 - H_L}{1 - (H_L - \lambda)}, \quad \mu_\varphi = \mu_L \frac{\lambda}{(H_L - \lambda)} + \mu_g \frac{1 - H_L}{1 - (H_L - \lambda)} \quad \text{Re}_\varphi = \frac{G_\varphi D_{\text{eff}}}{\mu_\varphi}$$

$$\frac{1}{\sqrt{f_\varphi}} = -2 \log \left[\frac{2\varepsilon}{D_{\text{eff}}} + \frac{18.7}{\text{Re}_\varphi \sqrt{f_\varphi}} \right] + 1.74$$

¹H_L calculated using (Mukherjee and Brill)

²H_L calculated using (Beggs and Brill)

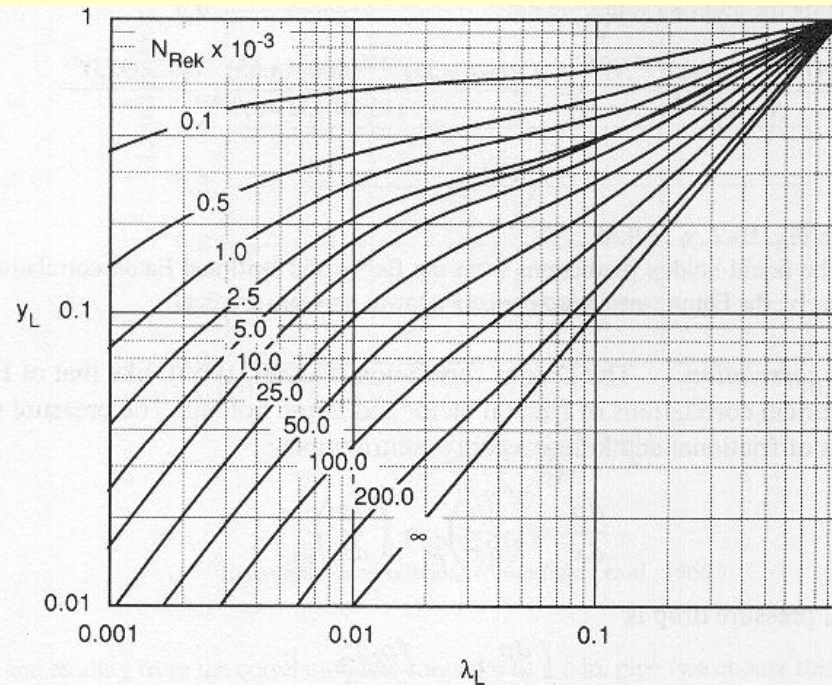
2. Continued

Dukler(1969)

$$-\left(\frac{dP}{dz}\right)_f = \frac{f\rho_k v_m^2}{2d_{\text{eff}}} \quad \text{where } \rho_k = \frac{\rho_L \lambda_L^2}{H_L} + \frac{\rho_g \lambda_g^2}{H_g}, \quad \lambda_L = \frac{v_{SL}}{v_m}, \quad \lambda_g = \frac{v_{Sg}}{v_m}$$

$$\mu_k = \mu_L \lambda_L + \mu_g \lambda_g, \quad \text{Re}_k = \frac{\rho_k v_m d}{\mu_k}, \quad y = -\ln(\lambda_L) \quad \text{and} \quad f_k = 0.0056 + 0.5 \text{Re}_k^{-0.32}$$

$$\frac{f}{f_k} = 1 + \frac{y}{1.281 - 0.478y + 0.444y^2 - 0.094y^3 + 0.0084y^4}$$



Outlet pressure calculated and measured

Correlation	Mean% error	Abs Error	Std. Dev.	NSME	Roughness (μm)
B&B (1973)	6.24	20.16	212.98	0.102	950
M&B (1985)	4.77	17.98	193.57	0.0864	1335
O¹(1976)	5.02	20.69	229.72	0.1325	980
O²(1976)	3.0	18.0	188.93	0.1325	980
D(1969)	-27.36	27.36	225.49	0.0638	>2500

Outlet pressure calculated and measured

Correlation	Mean% error	Abs Error	Std. Dev.	NSME	Equivalent Diameter
B&B (1973)	-8.25	12.45	190.68	0.0542	0.605
M&B (1985)	4.06	12.64	207.24	0.0728	0.509
O¹(1976)	10.86	15.14	204.63	0.061	0.604
O²(1976)	5.42	10.67	195.59	0.0655	0.567
D(1969)	12.9	17.44	196.31	0.0707	0.538

Conclusion remarks

- Flash calculations were performed using the process simulator Hysys version 3.1 based on PVT data obtained from the PVT laboratory in PRC in KISR
- Flow through the pipeline was simulated using a comprehensive computer program written using the C++. The program calculated outlet pressure based on inlet pressure, volumetric flow, thermo-physical properties, size of pipeline and its elevation as a function of distance along its length.
 - Five most commonly used correlations were used to evaluate surface roughness in the pipe.
 - The same exercise was repeated fixing the roughness value and determining equivalent diameter as pipeline had particle deposition.

Conclusion remarks

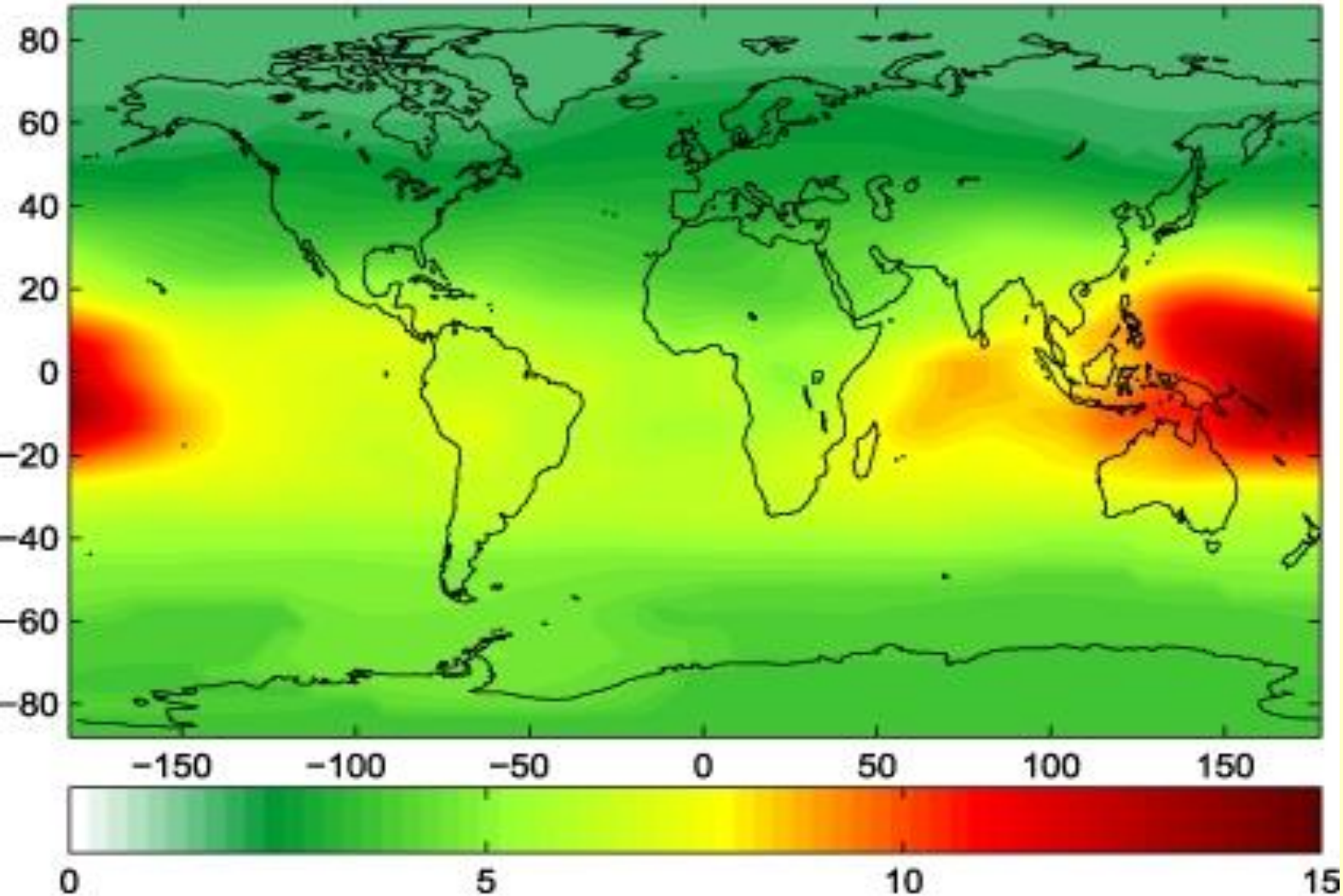
- All correlation presented that single-phase flow occurred initially within the pipeline due to high inlet pressure. The onset of two phase flow was predicted to occur within the second half of the 42km pipeline.
- Two approaches were adopted, first one determining pipe-roughness was used as fitting parameter. The obtained values for 30 different flow conditions were 20 to 50 times the published value for new wrought iron pipe.
- In second approach, the possibility of partial blockage by deposition of sand and silt was explored. The results showed 8.1% to 14.3% solid holdup that was somehow satisfactory as the pipeline has been laid before 30 years.

Conclusion remarks

- **Both methods indicated that Oliemans (1976) correlation with liquid holdup calculated using Beggs and Brill(1973) has provided the best overall fit to the entire data.**

NO_x-related O₃ impact from Temporal and spatial variability in the aviation Industry

O₃ Sensitivity to NO_x Emissions at Cruise Altitudes



Gilmore, C.K., Barrett, S.R.H., Koo, J and Wang, O. , “Temporal and spatial variability in the aviation NO_x-related O₃ impact”, *Environ. Res. Lett.* **8**:(3) doi:10.1088/1748-9326/8/3/034027

Figure 3 from Temporal and spatial variability in the aviation NO_x-related O₃ impact

